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(54) **SYSTEM FOR CALIBRATION OF A COMPRESSOR UNIT IN A HEATING, VENTILATION, AND AIR CONDITIONING SYSTEM**

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(58) **Field of Classification Search**

USPC 73/1.57
See application file for complete search history.

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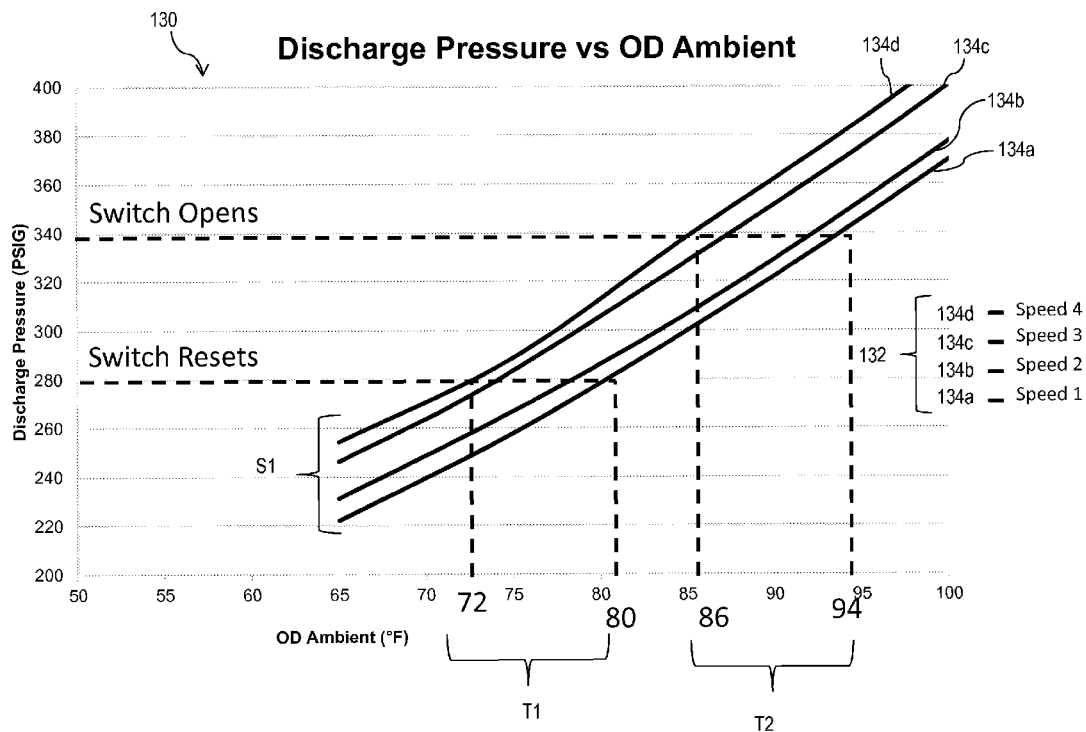
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(57) **ABSTRACT**

The present invention provides for a system for calibrating operation of a compressor unit in a heating, ventilation, and air conditioning (HVAC) system. A measuring device measures an operating parameter of the HVAC system at a position where the measuring device is mounted on a refrigerant line of the HVAC system. The measuring device switches states when the value of the measured operating parameter reaches a switching value. A controller estimates a value of the first operating parameter at the position where the first measuring device is mounted on the refrigerant line, and the controller determines whether the estimated first operating parameter is within a threshold percentage of the switching value.

20 Claims, 4 Drawing Sheets



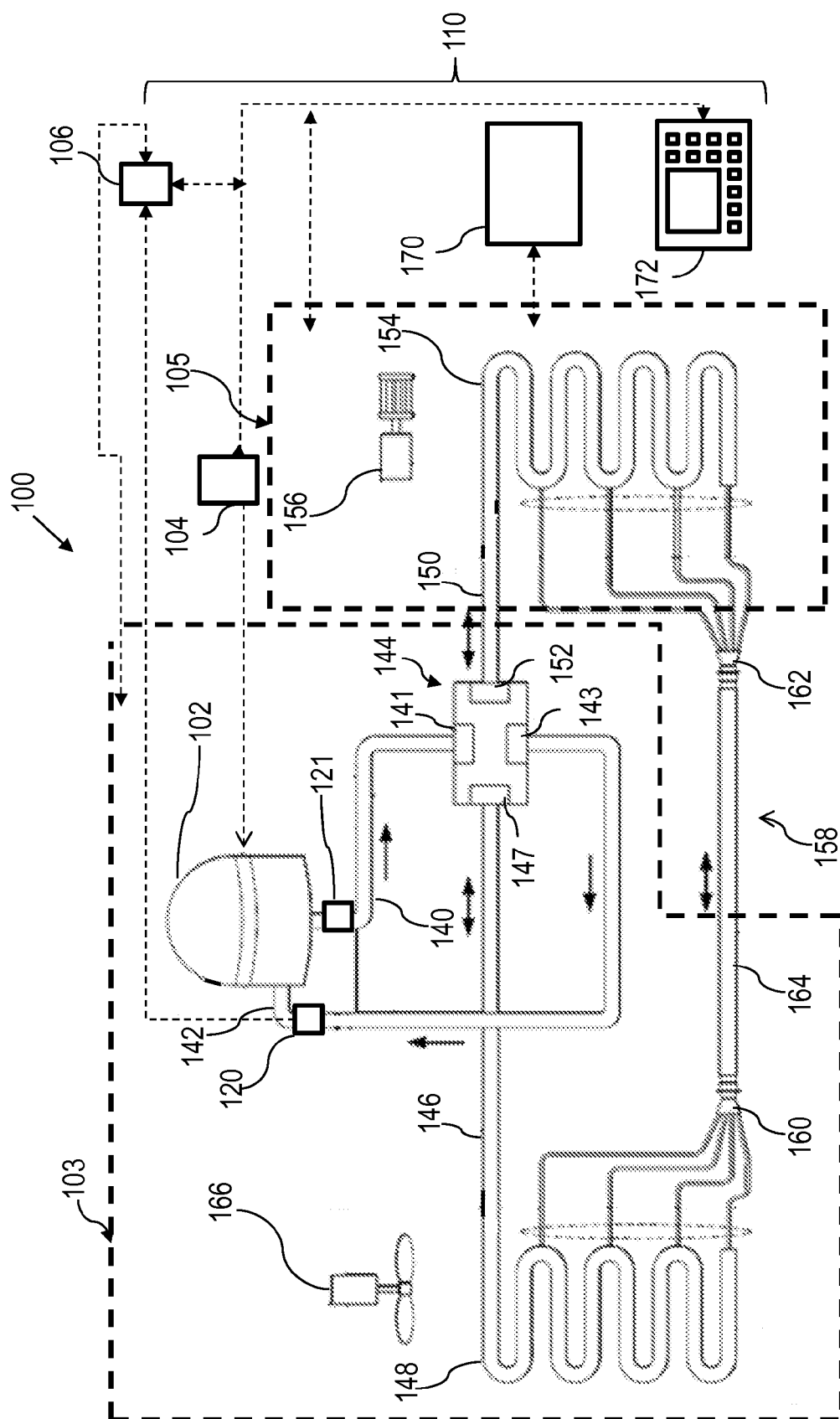


FIG. 1

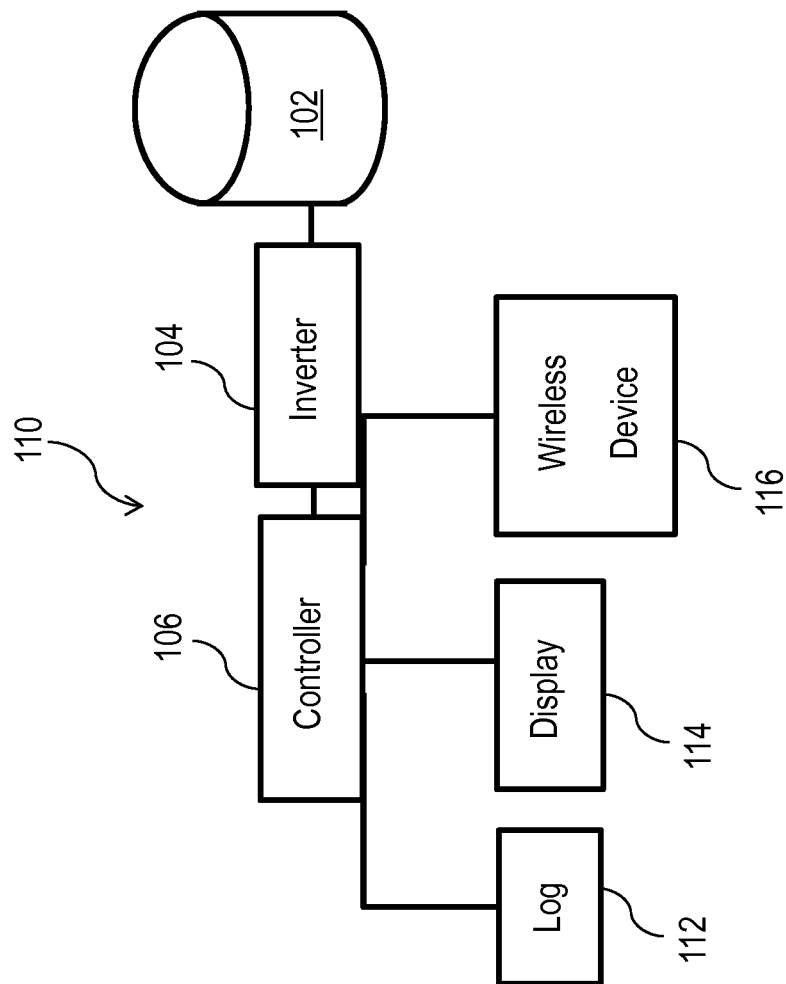
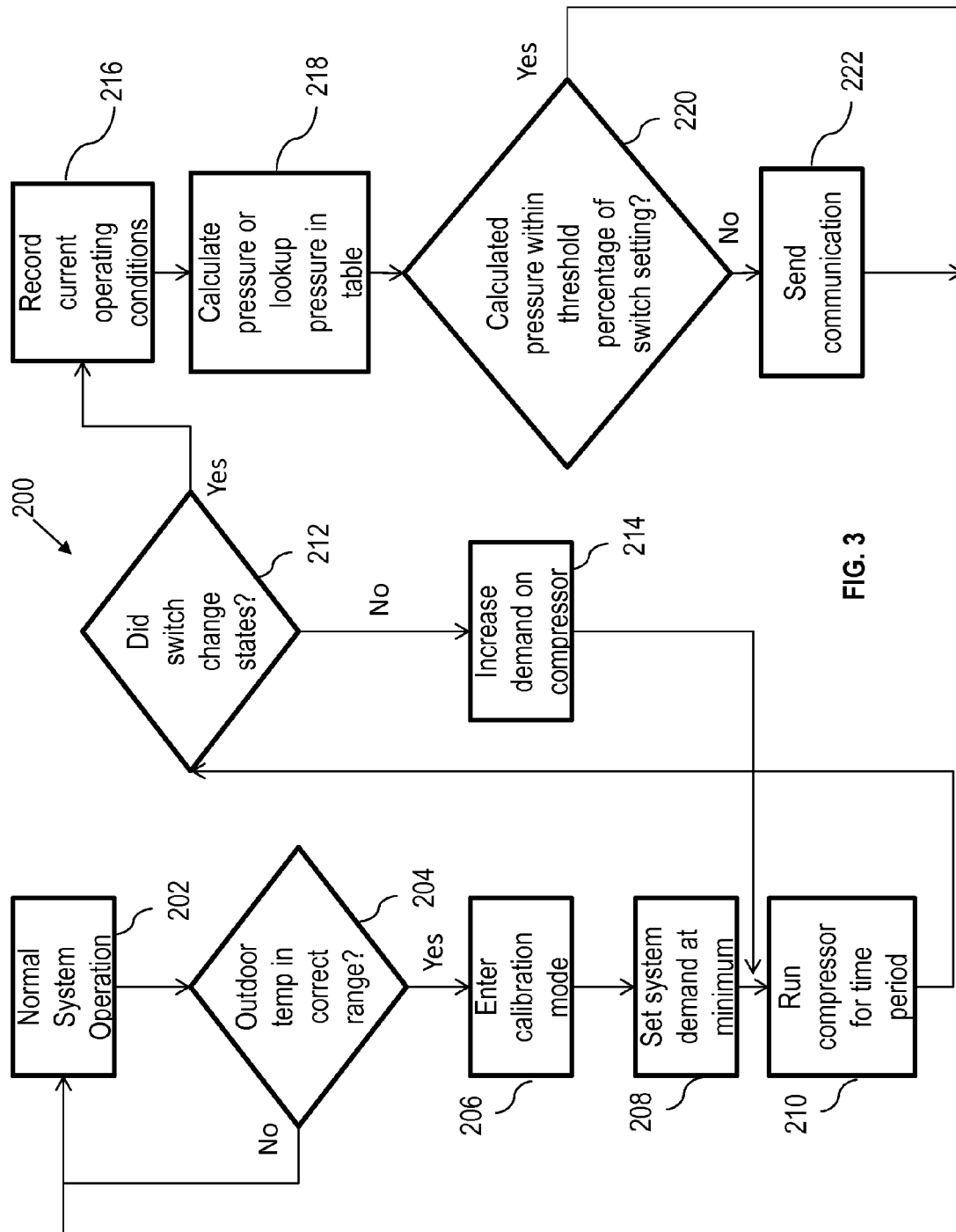


FIG. 2



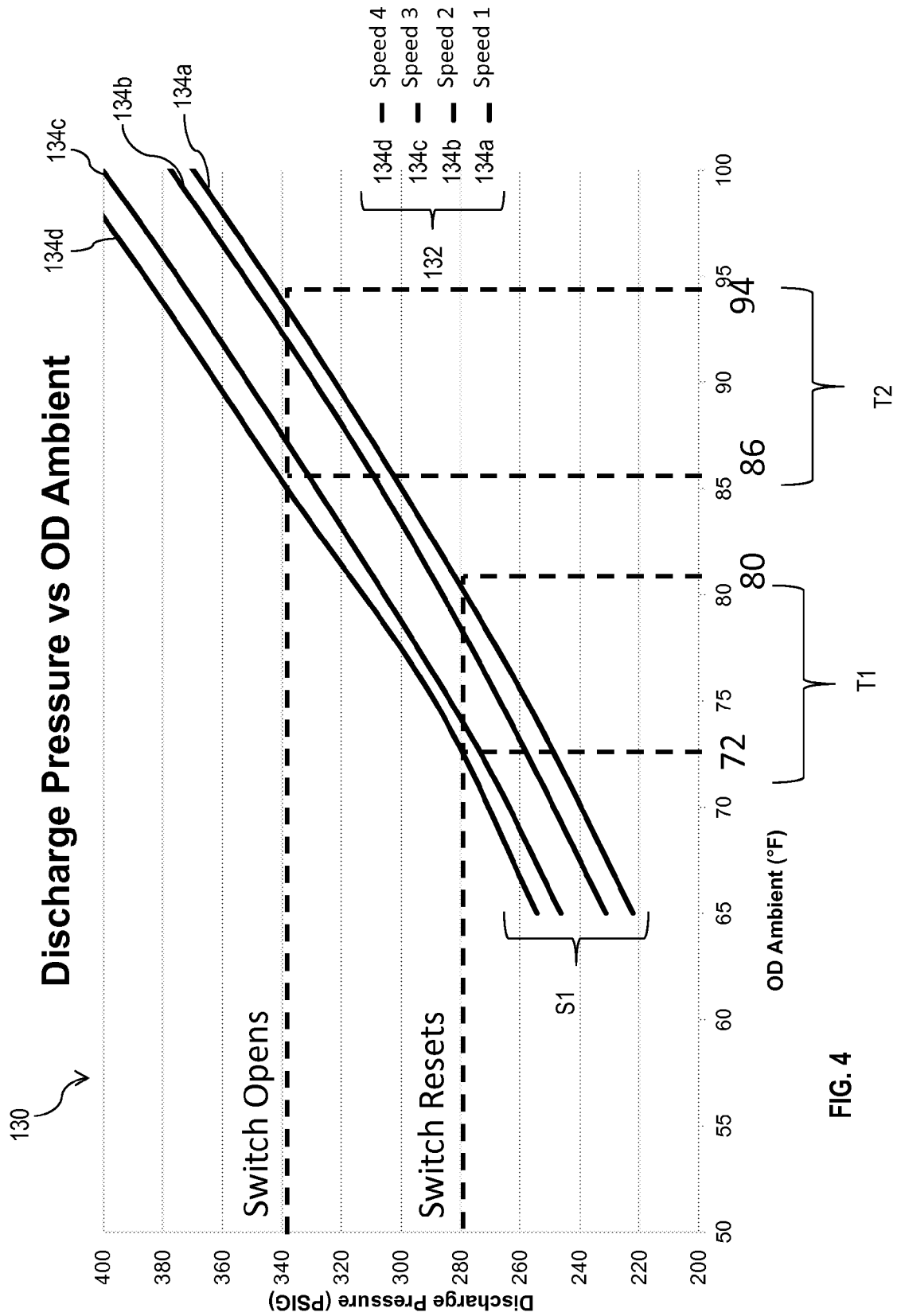


FIG. 4

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SYSTEM FOR CALIBRATION OF A COMPRESSOR UNIT IN A HEATING, VENTILATION, AND AIR CONDITIONING SYSTEM

CROSS-REFERENCED APPLICATIONS

This application relates to co-pending U.S. patent application Ser. No. 14/173,686, entitled SYSTEM FOR CONTROLLING OPERATION OF AN HVAC SYSTEM, filed Feb. 5, 2014.

BACKGROUND OF THE INVENTION

Field of the Invention

The present invention relates to calibration systems used in heating, ventilation, and air conditioning (HVAC) systems and, more particularly, to a system for calibrating operation of a compressor unit in an HVAC system.

The discharge pressure and suction pressure of a compressor in a heating, ventilation, and air conditioning (HVAC) system can be used as a diagnostic tool to troubleshoot problems with the system or to confirm that the system is operating normally. A model of the discharge pressure and suction pressure may be created from test data in a laboratory or manufacturing facility.

Once the HVAC system is installed, however, use of discharge pressure and suction pressure as a diagnostic tool is time consuming and expensive. Trained technicians may manually determine and utilize discharge and suction pressure in diagnostics. A pressure transducer may also be utilized as a tool for determining these pressures. Because of time and costs involved, suction and discharge pressure cannot often be used in the diagnostic process. What is needed are systems and methods for utilizing suction and discharge pressure as diagnostic tools in the calibration of HVAC systems.

SUMMARY

The present invention provides for a system for calibrating operation of a compressor unit in a heating, ventilation, and air conditioning (HVAC) system. A controller estimates the value of an operating parameter of the HVAC system at a position on a refrigerant line of the HVAC system where a measuring device is mounted to measure the operating parameter. The controller determines whether the estimated operating parameter is within a threshold percentage of the value of the operating parameter at which the measuring device switches states.

BRIEF DESCRIPTION OF THE DRAWINGS

For a more complete understanding of the present invention and the advantages thereof, reference is now made to the following Detailed Description taken in conjunction with the accompanying drawings, in which:

FIG. 1 illustrates an HVAC system;

FIG. 2 illustrates a compressor assembly and a control assembly;

FIG. 3 shows a flow chart of steps in a method for calibrating operation of a compressor unit in an HVAC system; and

FIG. 4 shows a graph of discharge pressure of a compressor unit versus outside temperature.

DETAILED DESCRIPTION

In the following discussion, numerous specific details are set forth to provide a thorough understanding of the present

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invention. However, those skilled in the art will appreciate that the present invention may be practiced without such specific details. In other instances, well-known elements have been illustrated in schematic or block diagram form in order not to obscure the present invention in unnecessary detail. Additionally, for the most part, details concerning well-known features and elements have been omitted inasmuch as such details are not considered necessary to obtain a complete understanding of the present invention, and are considered to be within the understanding of persons of ordinary skill in the relevant art.

Referring to FIG. 1, a first compressor unit **102** may be configured to operate in a heat pump, such as a heat pump system **100**. The first compressor unit **102** may be configured to compress low pressure refrigerant in vapor form to a higher pressure vapor. It will be understood by persons of ordinary skill that the first compressor unit **102** may operate within the heat pump system **100** in conjunction with other known components.

The heat pump system **100** may comprise an outdoor unit **103** operatively connected to an indoor unit **105**. The outdoor unit **103** may comprise the first compressor unit **102** having a suction line **142** and a discharge line **140**. The suction line **142** and the discharge line **140** may operatively connect the first compressor unit **102** with a reversing valve **144** at an output port **143** and an input port **141**, respectively. The reversing valve **144** may be configured to change the direction of flow of refrigerant within the heat pump system **100**.

A first transfer line **146** may operatively connect a first reversing port **147** with an outside heat exchanger unit **148** configured to condense high pressure vapor refrigerant to a liquid. An outdoor fan **166** may be configured to blow outside air over the heat exchange section of the outside heat exchanger unit **148**.

A distributor **158** may be operatively connected to the outlet of the outside heat exchanger unit **148** at a first end **160** and operatively connected to the inlet of an outside heater exchanger unit **148** at a second end **162**. Liquid tubing **164** may span the first end **160** and the second end **162** for transporting substantially liquid refrigerant from the outside heat exchanger unit **148** to an inside heat exchanger unit **154**. The distributor **158** may function as a metering device to regulate the amount of liquid refrigerant flowing into the inside heater exchanger unit **154**.

The inside heat exchanger unit **154** may be configured to evaporate liquid refrigerant to a vapor. An indoor blower **156** may be configured to pull air over coils of the inside heat exchanger unit **154** and to circulate air into the enclosed space. A second transfer line **150** may operatively connect a second reversing port **152** with the inside heat exchanger unit **154** to transfer refrigerant in low pressure vapor form back into the compressor unit **102** through the suction line **142**.

It will be understood by persons of ordinary skill that operation of the heat pump system **100** may be changed from heating to cooling by reversing flow within the system so that the inside heat exchanger unit **154** operates as a condenser and the outside heat exchanger unit operates as an evaporator.

Referring to FIG. 2, a control assembly **110** may be operationally coupled to the first compressor unit **102**. The control assembly **110** may comprise an electrical power converter, such as a first inverter **104** and an electronic first controller **106**. The first inverter **104** may be operationally connected to the first compressor unit **102** and configured to adjust the input voltage delivered to the first compressor unit **102**.

The first inverter **104** may be operationally connected to a first controller **106** configured to receive and send operation

signals for operation of the HVAC system **100**. The first controller **106** may be an outside unit controller.

The first inverter **104** may be operationally coupled to the first compressor unit **102** and configured to modify the input voltage delivered to the first compressor unit **102**. The first inverter **104** may be operationally coupled to a first controller **106** configured to receive and send control signals for operation of the HVAC system **100**.

A compressor speed of the first compressor unit **102** may be controlled by the first inverter **104** receiving control signals from the first controller **106**. It will be understood that the compressor speed may be controlled without the use of an inverter as part of the control assembly **110**. In those embodiments, the first controller **106** may be configured to utilize other known variable-speed solutions, including but not limited to pulse width modulation of the compression process, or modulating the capacity by bypassing some of the refrigerant around the compression process (referred to as “unloading”).

The control assembly **110** may further comprise other control devices, such as an inside control unit **170** and a thermostat **172**. The inside control unit **170** may operate in conjunction with the first controller **106** to control operation of the indoor blower **156** and compressor unit **102**. The thermostat **172** may adjust the target temperature and have other commonly-used functions to control the environmental conditions of the enclosed space.

Referring to FIG. 1, the HVAC system **100** may further comprise a measuring device configured to measure an operating parameter of the HVAC system **100**. In some embodiments, the measuring device comprises a pressure switch **121** mounted on the discharge line **140** configured to monitor the discharge pressure of the first compressor unit **102**. The first controller **106** may be operationally connected to the pressure switch **121** to control the monitoring process and store data.

The pressure switch **121** may comprise an actuation pressure and a reset pressure. The actuation pressure is the discharge pressure at which the switch opens or closes from a normal position, and the reset pressure is the discharge pressure at which the switch returns to a normal position. It will be understood by persons of ordinary skill in the art that the pressure switch **121** may be configured to monitor other pressure values or operating parameters in the HVAC system that may be useful in the systems and methods disclosed in this description, including but not limited to the suction pressure.

Referring to FIGS. 3 and 4, the first controller **106** may be configured to perform one or more methods for calibrating operation of the HVAC system **100**. The HVAC system **100** may be configured to operate in a calibration mode. Calibration of the HVAC system **100** may comprise a comparison between a normal operation model **130** of the HVAC system **100** and a measured operating parameter of the HVAC system **100**. The model **130** may characterize normal operation as a function of one or more operating conditions. The model **130** shown in FIG. 4 was prepared in laboratory tests using an HVAC system having a variable capacity compressor unit of 5 tons. It will be understood by persons of ordinary skill in the art that a model of normal operation may be prepared for different types of HVAC systems having one or more compressor units of different capacities.

In some embodiments, as shown in FIG. 4, the model **130** may comprise a characterization of the discharge pressure of the HVAC system **100** as a function of the outside ambient temperature in which the compressor unit **102** operates. The model **130** may be compared to a measured value of discharge pressure. The comparison may be used for diagnostic purposes or to determine or infer other useful information about the operation of the HVAC system **100**.

Referring to FIG. 4, a normal operation model **130** of discharge pressure as a function of the outside ambient pressure may comprise a data set **132**. The data set **132** may comprise one or more pressure curves **134a-d** which represent the discharge pressures of the HVAC system at a set of outside ambient temperatures expected under normal operating conditions for a specific compressor speed where Speed 4 is the maximum speed of the compressor, Speed 1 is the minimum Speed of the compressor and Speeds 2 and 3 are between Speeds 1 and 4. For example, in some embodiments, Speed 1 corresponding to curve **134d** may equal about 22 Hertz (Hz); Speed 2 corresponding to curve **134c** may equal about 34 Hz; Speed 3 corresponding to curve **134b** may equal about 46 Hz; and Speed 4 corresponding to curve **134a** may equal about 58 Hz. It will be understood that range of speed used to characterize the normal operation model **130** may vary according to the configuration and capacity of the compressor used The data set **132** may be stored in a memory of the first controller **106**.

It will be further understood that other measuring devices may be used in place of the pressure switch **121** to obtain measured operating parameters of the HVAC system **100** that would be useful in the calibration processes disclosed in this description. For example, a bimetal temperature switch may be operationally coupled to the HVAC system and configured to measure a condensing temperature of the outdoor coil. A data set for normal operation of the HVAC system **100** may be developed as function of at least the condensing temperature in order to allow for a comparison to be made between normal expected operation and the measured operating parameter.

Referring to FIG. 3, a method for controlling operation of an HVAC system may comprise a first step **202** of verifying normal operation of the system **100**. This operation may include verifying that the compressor unit **102** is currently on. In other embodiments, the HVAC system may be in setback or away mode, when demand on the compressor unit **102** is low.

In a second step **204**, the controller assembly may verify that the outdoor temperature is in a calibration range. A calibration range may comprise a range of temperatures for a given range of operating speeds of the compressor unit **102** at which the pressure switch **121** changes states. For example, as shown in FIG. 4, the pressure switch **121** may be configured to reset at a discharge pressure of about 280 pounds per square inch gauge (psig) in a first temperature range T1 for a first speed range S1 of the compressor unit **102**. The pressure switch **121** may be configured to open at a discharge pressure of about 340 psig in a second temperature range T2 for the speed range S1 of the compressor unit **102**.

In a third step **206**, the first controller **106** may initiate a calibration mode, based on the verification that the outdoor temperature is in the calibration range. If the temperature is not in the calibration range, then the HVAC system **100** may continue in normal operation (step **202**).

In a fourth step **208**, the first controller **106** may set the compressor unit **102** to operate at a first operating speed. For example, the compressor unit **102** may be operated at the Speed 4, which may correlate to a maximum demand on the compressor unit **102**. It will be understood by persons of ordinary skill in the art that the maximum demand will be compressor-specific and will vary based on the configuration and size of the compressor.

In a fifth step **210**, the compressor unit **102** may be operated for a first time period t_1 . The time period t_1 may comprise a pre-determined amount of time configured to increase the discharge pressure of the compressor unit **102** by a pre-selected amount. For example, the time period t_1 may comprise about 5 minutes, which may elevate the discharge pressure by

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about 127%, when the compressor unit **102** is operating at the Speed 2 and at an outside temperature of about 90° F.

The time period t_1 may be configured to change the discharge pressure so that the discharge pressure crosses the threshold at which pressure switch **121** changes state. For example, operating the compressor unit **102** at the Speed 2 for 5 minutes may result in the pressure switch **121** actuating.

In step **212**, the actuate event of the pressure switch **121** may be recorded in a memory of the first controller **106**. If the pressure switch **121** does not change states, the first controller **106** may generate a signal (step **214**) to the compressor unit **102** to change demand to operate the compressor unit at a second operating speed. The change in demand may comprise a pre-selected percentage of change from the initial demand load, where the pre-selected percentage is configured to change the discharge pressure of the compressor unit **102**. In some embodiments, the discharge pressure is elevated to change the state of the pressure switch **121**.

In step **210**, the compressor unit **102** may be operated for a second time period t_2 . The time period t_2 may comprise a pre-determined amount of time configured to change the discharge pressure of the compressor unit **102** by a pre-selected amount. It will be understood that the time periods t_1 and t_2 may be based on the expected operational characteristics of the HVAC system **100**, including but not limited to the compressor unit capacity.

In step **216**, the current operating state of the HVAC system at the actuate event (recorded in step **212**) may be recorded in a memory of the first controller **106**. The operating state may comprise known values of the HVAC system **100** and other data readily accessible through measurement by sensors or calculable based on known or measured data within the control assembly **110**, including data regarding operation of the compressor unit **102** taken from the inverter **104**.

In step **218**, the discharge pressure of the compressor unit **102** may be determined based on the operating state. For example, the discharge pressure may be calculated according to the systems and methods described in U.S. Application entitled "SYSTEM FOR CONTROLLING OPERATION OF AN HVAC SYSTEM", which is here incorporated by reference. In other embodiments, a data set containing a characterization of discharge pressure, for example in a table format, may be stored in memory and the predicted discharge pressure may be determined by correlating one or more parameters of the operating state of the compressor unit **102** with the stored estimated value of the predicted discharge pressure. Alternatively, the estimated discharge pressure may be received by the first controller **106** from a separate device, including but not limited to a pressure transducer operationally connected to the HVAC system.

In step **220**, the first controller **106** may compare the calculated discharge pressure of step **218** to the value of the pressure at which the pressure switch **121** is configured to change states (referred to as the "switch setting"). The first controller **106** may determine whether the calculated discharge pressure is within a threshold percentage of the switch setting. In some embodiments, the threshold percentage may be in a range of -10% to +10% of the switch setting. The threshold percentage may be pre-selected and programmed into the controller **106** based on the acceptable operational tolerances of the HVAC system **100**.

In step **222**, the first controller **106** may be configured to generate and send a communication based on whether the calculated discharge pressure is within the pre-selected percentage of the switch setting. For example, if the calculated discharge pressure is outside the threshold percentage of the

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switch setting, the first controller **106** may send a communication indicating that the HVAC system **100** is not operating within acceptable parameters.

The communication may comprise a textual or visual summary of data regarding operation of the HVAC system **100**, including a characterization of discharge pressure of the compressor unit **102**, such as a chart, graph, or table. The communication may be sent to a display, stored in memory, or communicated directly to a third party. Referring to FIG. **2**, the communication may be stored in a memory log **112** operationally connected to the first controller **106**. The predicted pressure may be sent to a display **114**. For example, a diagnostician may be connected to a port operationally connected to the controller **106** and may request a reading of the predicted discharge pressure, or may access the memory log **112** that contains a history of the predicted pressure for a given time period. In other embodiments, the communication generated by the first controller **106** in step **222** may be sent via a wireless device, for example as an email or text message.

In other embodiments, the measuring device, such as the pressure switch **121**, may be positioned at other positions within the HVAC system **100** for calibrating the pressure at that position, including but not limited to positioning a pressure switch **120** positioned on the suction line **142** for measuring suction pressure, as shown in FIG. **1**.

Having thus described the present invention by reference to certain of its preferred embodiments, it is noted that the embodiments disclosed are illustrative rather than limiting in nature and that a wide range of variations, modifications, changes, and substitutions are contemplated in the foregoing disclosure and, in some instances, some features of the present invention may be employed without a corresponding use of the other features. Many such variations and modifications may be considered desirable by those skilled in the art based upon a review of the foregoing description of preferred embodiments. Accordingly, it is appropriate that the appended claims be construed broadly and in a manner consistent with the scope of the invention.

The invention claimed is:

1. A system for calibrating operation of a compressor unit in a heating, ventilation, and air conditioning (HVAC) system, the system comprising:

a control assembly configured to operationally connect to a compressor assembly of an HVAC system, wherein the compressor assembly comprises a first compressor unit, wherein the first compressor unit operates at an outside ambient temperature, and wherein the controller assembly comprises:

a first measuring device mounted on a first line of the compressor unit, the first measuring device configured to switch states from a first state to a second state, wherein the first line comprises a channel for the flow of refrigerant with the HVAC system, wherein the first measuring device measures at least a first operating parameter of the HVAC system at the position on the first line where the first measuring device is mounted, and wherein the first measuring device switches from the first state to the second state when the value of the measured first operating parameter reaches a first switching value; and

a controller configured to estimate a first estimated value of the first operating parameter at the position where the first measuring device is mounted on the first line, and wherein the controller is configured to determine whether the first estimated value is within a first threshold percentage of the first switching value.

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2. The system of claim 1, wherein the controller is configured to generate a first communication based on the determination of whether the first estimated value is within the first threshold percentage of the first switching value.

3. The system of claim 2, wherein the controller is configured to send the first communication by a wireless connection with an external device.

4. The system of claim 2, wherein the controller is configured to determine whether the first measuring device has switched states from the first state to the second state.

5. The system of claim 4, wherein the controller is configured to change demand on the compressor unit from a first load to a second load if the first measuring device has not switched states from the first state to the second state.

6. The system of claim 5, wherein changing the demand on the compressor unit comprises changing a speed of the compressor unit.

7. The system of claim 5, wherein the controller is configured to initiate a first mode of operation for calibrating operation of the compressor unit, wherein the initiation of the first mode of operation is based on the outside ambient temperature, and wherein the controller is configured to operate the compressor unit under the second load for a first time period to change the measured first operating parameter of the HVAC system to cause the first measuring device to switch states.

8. The system of claim 7, wherein the controller is configured to determine whether the outside ambient temperature is within a first calibration range, and wherein the measuring device is configured to change states when the compressor unit is operated within the first calibration range.

9. The system of claim 8, wherein the first measuring device comprises a pressure switch and the first operating parameter comprises the pressure of the refrigerant in the first line at the position where the first measuring device is mounted, and wherein the pressure switch is configured to measure the pressure of the refrigerant at the position where the pressure switch is mounted on the first line.

10. The system of claim 8, wherein the first measuring device comprises a temperature switch and the first operating parameter comprises the temperature of the refrigerant in the first line at the position where the first measuring device is mounted, and wherein the temperature switch is configured to measure the temperature of the refrigerant at the position where the temperature switch is mounted on the first line.

11. A method for calibrating operation of a compressor unit in a heating, ventilation, and air conditioning (HVAC) system:

providing a control assembly configured to operationally connect to a compressor assembly of an HVAC system, wherein the compressor assembly comprises a first compressor unit configured to operate at an outside ambient temperature;

wherein the control assembly comprises a first measuring device mounted to a first line of the compressor unit, and wherein the first measuring device is configured to switch states from a first state to a second state at a first switching value, wherein the first line comprises a channel for the flow of refrigerant with the HVAC system, wherein the first measuring device measures at least a first operating parameter at the position on the first line where the first measuring device is mounted;

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switching, by the first measuring device, to the second state when the value of the measured first operating parameter reaches the first switching value;

estimating, by the controller, a first estimated value of the first operating parameter at the position where the first measuring device is mounted on the first line; and determining, by the controller, whether the first estimated value is within a first threshold percentage of the first switching value.

12. The method of claim 11, further comprising:

generating, by the controller, a first communication based on the determination of whether the first estimated value is within the first threshold percentage of the first switching value.

13. The method of claim 12, further comprising sending, by the controller, the first communication by a wireless connection with an external device.

14. The method of claim 12, further comprising determining whether the first measuring device has switched states from the first state to the second state.

15. The method of claim 14, further comprising changing demand on the compressor unit from a first load to a second load if the first measuring device has not switched states from the first state to the second state.

16. The method of claim 15, wherein changing the demand on the compressor unit comprises changing a speed of the compressor unit.

17. The method of claim 15, further comprising:

initiating, by the controller, a first mode of operation for calibrating operation of the compressor unit, wherein the initiation of the first mode of operation is based on the outside ambient temperature; and

operating, by the controller, the compressor unit under the second load for a first time period to change the measured first operating parameter at the first line from a first measured value to a second measured value, wherein the change in the first operating parameter at the first line from the first measured value to the second measured value of the first operating parameter causes the first measuring device to switch states.

18. The method of claim 17, further comprising:

measuring, by the controller, the outdoor temperature; and changing, by the first measuring device, states when the compressor unit is operated at the outside ambient temperature within a first calibration range.

19. The method of claim 17, wherein the first measuring device comprises a pressure switch, and the first operating parameter comprises the pressure of the refrigerant in the first line at the position where the first measuring device is mounted, and wherein the pressure switch is configured to measure the pressure of the refrigerant at the position where the pressure switch is mounted on the first line.

20. The method of claim 17, wherein the first measuring device comprises a temperature switch and the first operating parameter comprises the temperature of the refrigerant in the first line at the position where the first measuring device is mounted, and wherein the temperature switch is configured to measure the temperature of the refrigerant at the position where the temperature switch is mounted on the first line.

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